

# Contemporary Challenge of Connecting Turbulence Models with Time and Spectral Analytical Acoustic Sources

74th American Physical Society Annual Meeting of the  
Division of Fluid Dynamics

Steven A. E. Miller  
Assistant Professor  
University of Florida

Department of Mechanical and Aerospace Engineering  
Theoretical Fluid Dynamics and Turbulence Group

# Outline

- Introduction
- Typical RANS models for Aeroacoustics
- Traditional Connection
- LES / FW-H Approach and its Major Drawbacks
- Analytical Acoustic Sources from Turbulence
  - Small-Scale Turbulence
  - Shockwave Shear-Layer Interaction
  - Large-Scale Coherent Turbulence, an enigma in the acoustics community
- Connection to Wavenumber Spectra
  - Noise is the fingerprint of turbulence
- Disconnect Between Turbulence Closure and Aeroacoustics
- Summary

# Introduction

- The turbulence modeling community and the aeroacoustics community generally create separate prediction models
  - Turbulence modeling community goal is to predict accurate turbulent flow statistics
  - Acoustic community wishes to predict accurate radiated noise
- Outcome of both are relatable through semi-empirical means
  - Usually up to acoustics community to make the connection
- Approach of LES can be used with Ffowcs-Williams and Hawking approach
  - We learn nothing about the sources of noise and the sources of FWH are analogies of the actual source of turbulent noise
  - Huge computational expense and does not make use of typical large RANS databases available within industry

# RANS I

Reynolds-averaged Navier-Stokes equation (RANS)

$$\rho \frac{\partial U_i}{\partial t} + \rho U_j \frac{\partial U_i}{\partial x_j} = -\frac{\partial P}{\partial x_i} + \frac{\partial}{\partial x_j} \left( 2\mu S_{ji} - \overline{\rho u'_j u'_i} \right) \quad (1)$$

Reynolds stress equation

$$\begin{aligned} \frac{D\overline{u_i u_j}}{Dt} = \frac{\partial}{\partial X_l} \left( -\overline{u_i u_j u_l} - \frac{p}{\rho} (\delta_{jl} u_i + \delta_{il} u_j) + \nu \frac{\partial \overline{u_i u_j}}{\partial X_l} \right) \\ - \left( \overline{u_i u_l} \frac{\partial U_j}{\partial X_l} + \overline{u_j u_l} \frac{\partial U_i}{\partial X_l} \right) - 2\nu \overline{\frac{\partial u_i}{\partial X_l} \frac{\partial u_j}{\partial X_l}} + \frac{p}{\rho} \left( \frac{\partial u_i}{\partial X_j} + \frac{\partial u_j}{\partial X_i} \right) \end{aligned} \quad (2)$$

# RANS I

$\epsilon$  equation is

$$\begin{aligned} \frac{D\epsilon}{Dt} = \frac{\partial}{\partial X_l} \left( -\overline{\epsilon' u_l} - \frac{2\nu}{\rho} \overline{\frac{\partial u_l}{\partial X_j} \frac{\partial p}{\partial X_j}} + \nu \frac{\partial \epsilon}{\partial X_l} \right) - 2\nu \overline{u_l \frac{\partial u_l}{\partial X_j} \frac{\partial^2 U_i}{\partial X_i \partial X_j}} \\ - 2\nu \frac{\partial U_i}{\partial X_j} \left( \overline{\frac{\partial u_l}{\partial X_i} \frac{\partial u_l}{\partial X_j}} + \overline{\frac{\partial u_l}{\partial X_l} \frac{\partial u_j}{\partial X_l}} \right) \\ - 2\nu \overline{\frac{\partial u_i}{\partial X_j} \frac{\partial u_i}{\partial X_l} \frac{\partial u_j}{\partial X_l}} - 2 \overline{\left( \nu \frac{\partial^2 u_i}{\partial X_l \partial X_l} \right)^2} \end{aligned} \quad (3)$$

TKE equation of the mean motion

$$\frac{D}{Dt} \left( \overline{\frac{\tilde{u}_i \tilde{u}_i}{2}} \right) = -\tilde{u}_i \frac{\partial \bar{p}}{\partial x_i} + \tilde{u}_i \frac{\partial \bar{\tau}_{ik}}{\partial x_k} - \tilde{u}_i \frac{\partial}{\partial x_k} \left( \overline{\rho u'_i u'_k} \right) \quad (4)$$

# Outcome of RANS CFD Simulations

## Typical RANS Output

- Geometry
- Computational grid
- Mean variables
- Two-equation models
  - Usually  $k$  and  $\omega$  or  $\epsilon$
- Reynolds Stress Models – Reynolds stresses and  $\epsilon$  or  $\omega$

# Acoustic Analogy – A Traditional Path via Lighthill

- Rearrangement of continuity and momentum into wave equation with unknown left and right hand sides
- Sources are analogues of actual sources, which is why it is called the acoustic analogy

$$\frac{\partial^2 \rho}{\partial t^2} - c_\infty^2 \nabla^2 \rho = \frac{\partial^2 T_{ij}}{\partial x_i \partial x_j} \quad (5)$$

and

$$T_{ij} = \rho v_i v_j + p_{ij} - c_\infty^2 \rho \delta_{ij} \quad (6)$$



Figure 1: Sir James Lighthill.

# Traditional Way of Connecting Acoustic Sources to RANS

Acoustic source models depend on scales to estimate two-point correlations

Length scale

$$l_x(\mathbf{y}) = c_l K(\mathbf{y})^{3/2} / \epsilon(\mathbf{y}) \quad (7)$$

Time scale

$$\tau_s(\mathbf{y}) = c_\tau K(\mathbf{y}) / \epsilon(\mathbf{y}) \quad (8)$$

Velocity scale

$$u_s(\mathbf{y}) = u_s \sqrt{2K(\mathbf{y})/3} \quad (9)$$



## Connecting Lighthill's Model to RANS

$$\frac{\overline{\partial^2 T_{ij} \partial^2 T'_{lm}}}{\partial \tau^2 \partial \tau'^2} \approx \frac{\partial^4 \mathbf{R}_{ijlm}}{\partial \tau^4}. \quad (10)$$

and

$$\mathbf{R}_{ijlm} = \bar{\rho} \bar{\rho}' \left( \overline{u_i u_j} \overline{u'_l u'_m} \right) R, \quad (11)$$

The normalized two-point cross-correlation,  $R$ , is modeled as

$$R = \exp \left[ -\frac{(\xi - \bar{u}\tau)^2}{l_{sx}^2} \right] \exp \left[ -\frac{(1 - \tanh[\alpha|\xi|])|\xi - \bar{u}\tau|}{l_{sx}} \right] \times \exp \left[ -\frac{|\xi|}{l_{sx}} \right] \exp \left[ -\frac{|\eta|}{l_{sy}} \right] \exp \left[ -\frac{|\zeta|}{l_{sz}} \right], \quad (12)$$

# Acoustic Prediction Models

Time-domain model for  $\rho$ ,  $\mathbf{u}$ , and  $p$

$$q_k^\perp(\mathbf{x}, t) = \int_{-\infty}^{\infty} \dots \int_{-\infty}^{\infty} \sum_{n=0}^4 \mathbf{q}_{g,k}^{\perp n}(\mathbf{x}, t; \mathbf{y}, \tau) \Theta_n(\mathbf{y}, \tau) d\tau d\mathbf{y} \quad (13)$$

and in the spectral domain for same field-variables

$$S_k^\perp(\mathbf{x}, \omega) = \int_{-\infty}^{\infty} \dots \int_{-\infty}^{\infty} \sum_{m=0}^4 \sum_{n=0}^4 \hat{q}_{g,k}^{*\perp m}(\mathbf{x}; \mathbf{y}, \omega) \hat{q}_{g,k}^{\perp n}(\mathbf{x}; \mathbf{y} + \boldsymbol{\eta}, \omega) \times R_{m,n}^\perp(\mathbf{y}, \boldsymbol{\eta}, \tau) d\tau d\boldsymbol{\eta} d\mathbf{y}. \quad (14)$$

# Exact Sources for Noise from Turbulence

Continuity

$$\Theta_0 = -\frac{\partial \underline{\rho}}{\partial t} - \frac{\partial \underline{\rho u}_j}{\partial x_j}, \quad (15)$$

the momentum sources are

$$\Theta_i = -\frac{\partial}{\partial t} [\underline{\rho u}_i] - \frac{\partial}{\partial x_j} [\underline{\rho u}_i u_j] + \frac{\partial}{\partial x_j} \left\{ \mu \left[ \frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i} \right] \right\} - \frac{2}{3} \frac{\partial}{\partial x_j} \left\{ \mu \frac{\partial}{\partial x_k} u_k \right\} - \frac{\partial p}{\partial x_j} \delta_{ij}, \quad (16)$$

for  $i = 1$  to  $3$ , and the energy source is

$$\Theta_4 = -\frac{\partial p}{\partial t} - \frac{\gamma - 1}{2} \frac{\partial \underline{\rho u}_k u_k}{\partial t} - \gamma \frac{\partial \underline{u}_j p}{\partial x_j} - \frac{\gamma - 1}{2} \frac{\partial \underline{\rho u}_j u_k u_k}{\partial x_j} + (\gamma - 1) \frac{\partial}{\partial x_j} \left( \frac{c_p \mu}{Pr} \frac{\partial T}{\partial x_j} \right) + (\gamma - 1) \frac{\partial}{\partial x_j} \left[ \mu u_i \left( \frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i} \right) \right] - \frac{2}{3} \delta_{ij} \frac{\partial}{\partial x_j} \left[ \mu u_i \frac{\partial u_k}{\partial x_k} \right]. \quad (17)$$

# Noise Source Terms from Small-Scale Turbulence Written as Two-Point Correlation

Dominant sources of noise from small-scale turbulent structures within fully developed turbulence are written as two-point cross-correlations

$$R^{(8)} = \left\langle \frac{D\check{\rho}^{(1)}}{Dt}, \frac{D\check{\rho}^{(2)}}{Dt} \right\rangle \approx \frac{\bar{\rho}^2 K_s^2}{\tau_s^2} \check{R} \approx \bar{\rho}^2 K_s^2 \Omega^2 \check{R}, \quad (18a)$$

$$R^{(9)} = \left\langle \frac{D\check{\rho}^{(1)}}{Dt}, \frac{D\bar{\rho}^{(2)} \check{u}_k^{(2)} \check{u}_k^{(2)}}{Dt} \right\rangle \approx \frac{\bar{\rho}^2 K_s^2}{\tau_s^2} \check{R} \approx \bar{\rho}^2 K_s^2 \Omega^2 \check{R}, \quad (18b)$$

and

$$R^{(10)} = \left\langle \frac{D\bar{\rho}^{(1)} \check{u}_k^{(1)} \check{u}_k^{(1)}}{Dt}, \frac{D\bar{\rho}^{(2)} \check{u}_k^{(2)} \check{u}_k^{(2)}}{Dt} \right\rangle \approx \frac{\bar{\rho}^2 K_s^2}{\tau_s^2} \check{R} \approx \bar{\rho}^2 K_s^2 \Omega^2 \check{R}. \quad (18c)$$

# Noise Source Terms from Shockwave Shear Layer Interaction Written as Two-Point Correlation

A rare source term in the time-domain

$$\gamma \frac{\partial \hat{u}_j^{(1)} \bar{p}^{(1)}}{\partial y_j} \quad (19)$$

and spectral domain

$$\left\langle \gamma \frac{\partial \hat{u}_j^{(1)} \bar{p}^{(1)}}{\partial y_j}, \gamma \frac{\partial \hat{u}_m^{(2)} \bar{p}^{(2)}}{\partial y_m} \right\rangle \quad (20)$$

# Approximations of Scaling Terms for Small Scale Turbulent Noise – Example Prediction

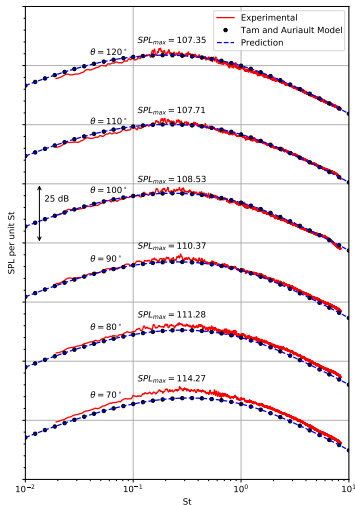


Figure 2: Prediction of fine-scale mixing noise at NPR = 1.893 and TTR = 3.2.

## A More Advanced Spectral Approach

$$\widehat{\langle \tilde{u}_i \rangle}(\mathbf{y}, \tau) = \sqrt{\frac{2}{3}} \left( \int_{\kappa_1}^{\kappa_2} E_u(\mathbf{y}, \kappa, \tau) d\kappa \right)^{1/2}, \quad (21)$$

$$\widehat{\langle \tilde{\rho} \rangle}(\mathbf{y}, \tau) = \left( \int_{\kappa_1}^{\kappa_2} E_\rho(\mathbf{y}, \kappa, \tau) d\kappa \right)^{1/2}, \quad (22)$$

$$\widehat{\langle \tilde{p} \rangle}(\mathbf{y}, \tau) = \left( \int_{\kappa_1}^{\kappa_2} E_p(\mathbf{y}, \kappa, \tau) d\kappa \right)^{1/2}, \quad (23)$$

and,

$$\widehat{\langle \tilde{T} \rangle}(\mathbf{y}, \tau) = \left( \int_{\kappa_1}^{\kappa_2} E_T(\mathbf{y}, \kappa, \tau) d\kappa \right)^{1/2}, \quad (24)$$

where  $E_u$ ,  $E_\rho$ ,  $E_p$ , and  $E_T$  are the 'energy' spectra of the difference functions of the field variables, and  $\kappa_1$  and  $\kappa_2$  are the limits of integration.

## Example Prediction from DNS Turbulence

Comparison of DNS with acoustic theory (no CFD) using wavenumber spectra from high-Re homogeneous isotropic turbulence

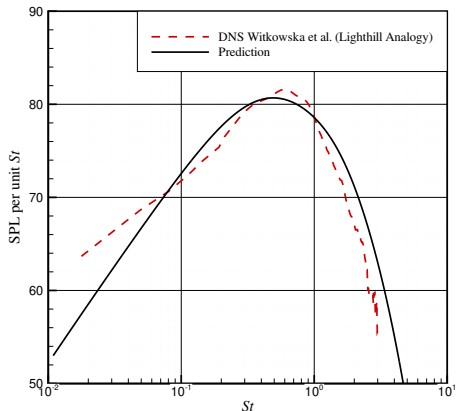


Figure 3: The power spectrum of acoustic pressure.



# Noise Model for Large-Scale Coherent Structures

- Closed form analytical models for the noise from large-scale structures are absent and being sought after in the community.
- One such skeleton for the model might look like...

$$S(\mathbf{x}, \omega) = \frac{\pi\omega^4}{c_\infty^4} g(\mathbf{x}, \omega) g^*(\mathbf{x}, \omega) \int_{-\infty}^{\infty} A_{ijlm} \frac{r_i r_j r_l r_m}{r^4} \frac{l_s l_{sy} l_{sz}}{\bar{u}} \exp\left[\frac{-l_s^2 \omega^2}{4\bar{u}^2}\right] \times \int_{-\infty}^{\infty} \exp\left[\frac{-i\xi\omega}{\bar{u}}\right] \exp\left[\frac{-|\xi|}{\bar{u}\tau_s}\right] d\xi dy_1 \quad (25)$$

Note appearance of typical RANS outcome and  $l_s$ . Model formulated this way on purpose to use RANS, intermittency and transient information is missing. Is it impossible to create a large-scale model from steady RANS?

# Contemporary Approaches

## RANS Closures versus Aeroacoustic Sources

### RANS

- Typically yields  $\bar{q}$  and  $k - \omega$  or  $\epsilon$
- Reynolds stresses are very helpful as have more direct connection with source models (e.g. Lighthill's model)

### Aeroacoustics

- Typically requires estimation of two-point correlations through model involving  $l$  and  $\tau$
- Requires calibration of additional empirical coefficients beyond RANS closure

# Desirable Characteristics

## RANS Closures versus Aeroacoustic Sources

### RANS

- Wavenumber spectra of field-variables
- Two-point correlations (is that possible?)
- Possible to form a RANS closure that yields aeroacoustic sources directly?

### Aeroacoustics

- Formulation of models for various sources in time and two-point correlation
- Formulation of a closed-form large-scale model
- Seamless integration with turbulence models at the same level of LES/FWH approach.
- Eliminate or reduce empirical coefficients

# Thank You